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Bacchetti

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(54) **HINGE FOR THE CONTROLLED ROTATABLE MOVEMENT OF A DOOR, IN PARTICULAR A GLASS DOOR**

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(51) **Int. Cl.**

E05D 5/02	(2006.01)
E05D 11/10	(2006.01)
E05D 3/02	(2006.01)
E05D 5/14	(2006.01)
E05D 11/08	(2006.01)

(52) **U.S. Cl.**

CPC ***E05D 11/1028*** (2013.01); ***E05D 3/02*** (2013.01); ***E05D 5/14*** (2013.01); ***E05D 11/08*** (2013.01); *E05D 11/1042* (2013.01); *E05D 2011/1035* (2013.01)

(58) **Field of Classification Search**

CPC E05D 5/0246; E05D 2005/0261; E05D 2005/0269; E05D 7/081; E05D 7/08; E05Y 2900/114; E05Y 2600/502; E05Y 2600/60; E05F 11/385; E06B 3/54; A47K 3/362; A47K 2003/367; A47F 3/12; A47F 3/125; Y10T 16/534; Y10T 16/5326; Y10T 16/554; Y10T 16/547

See application file for complete search history.

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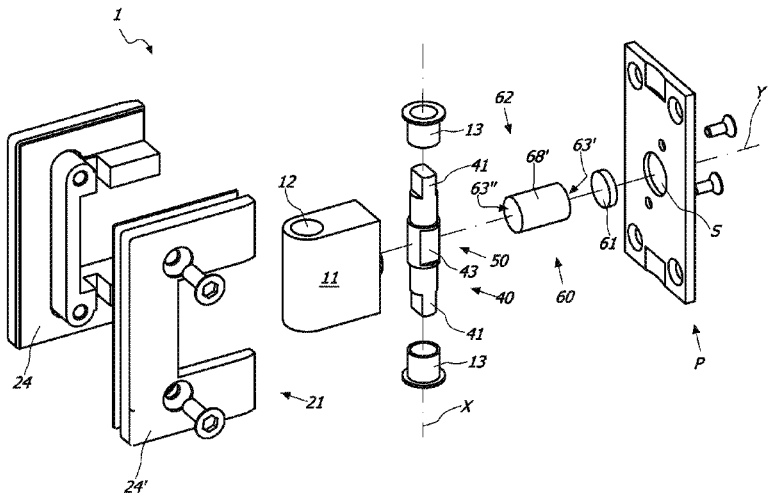
Primary Examiner — Chuck Mah

(74) *Attorney, Agent, or Firm* — Mark David Torche; Patwrite LLC

(57) **ABSTRACT**

A door hinge including a fixed element coupled to a wall; and a movable element coupled to the door. The fixed and movable elements are rotatably coupled to rotate between open and closed positions about a first axis. One of the movable and fixed elements includes a hinge body; the other of the movable and fixed elements includes a pivot and cam means. The hinge body includes a working chamber defining a second axis, which includes follower means interacting with the cam means to slide between first and second end-stroke positions. The cam means includes a flat face parallel to the first axis. The follower means includes an elastic element and an interface element having a first end interacting with the elastic element and a second end including a planar operating surface to come in contact with the flat face of the cam means to remain in contact and parallel.

13 Claims, 12 Drawing Sheets



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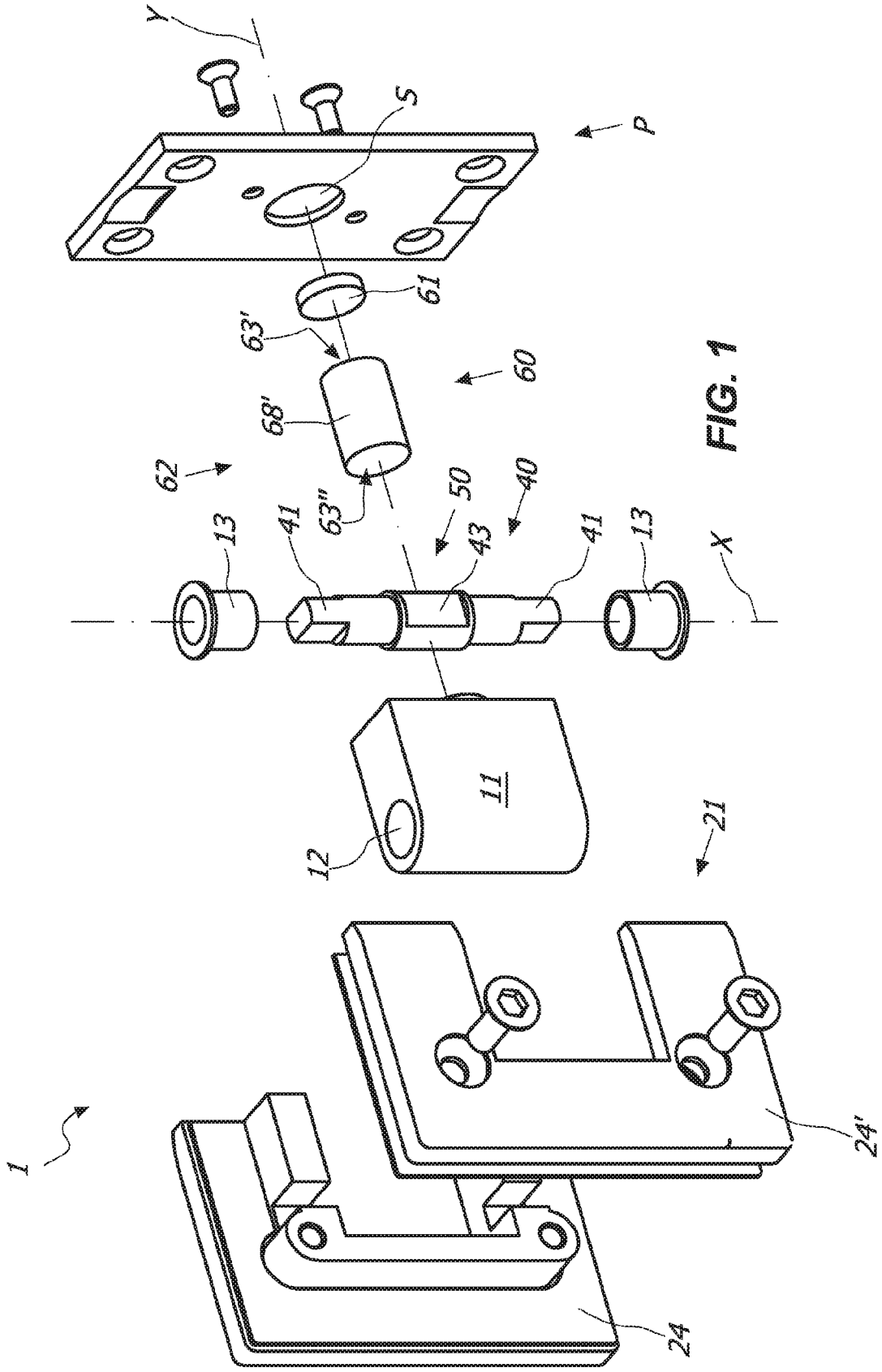

FIG. 1

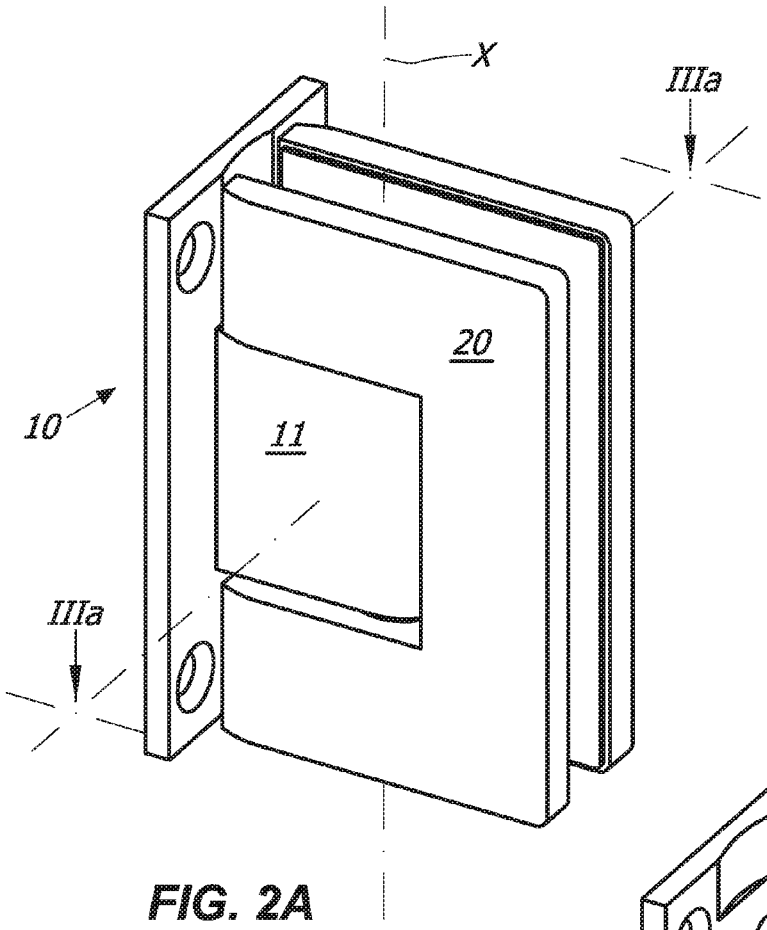

FIG. 2A

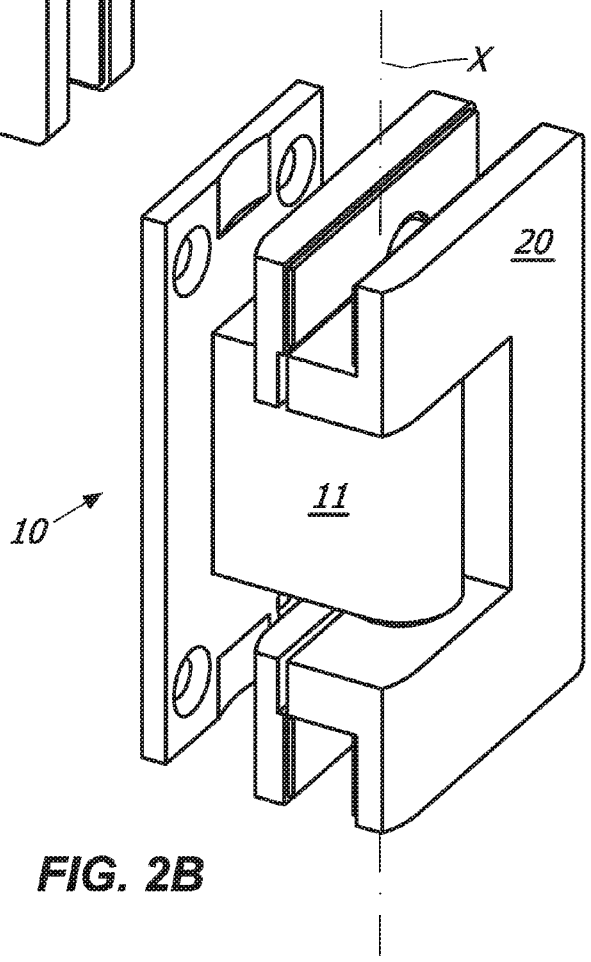

FIG. 2B

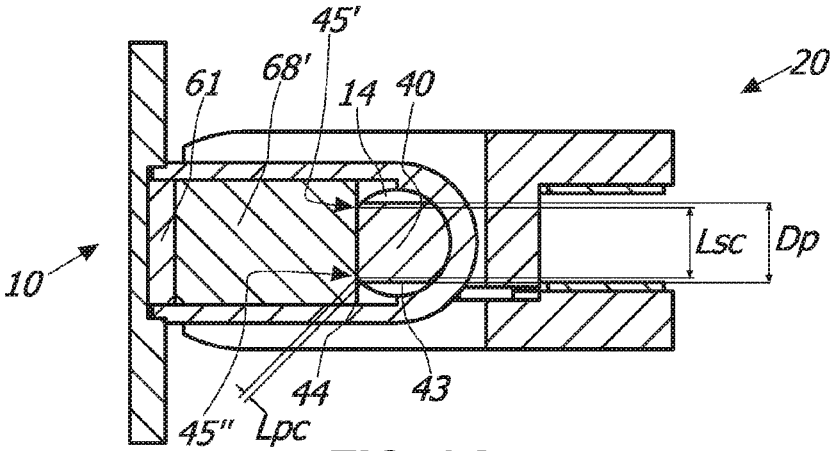

FIG. 3A

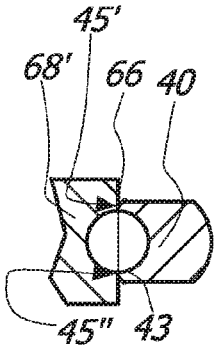

FIG. 9

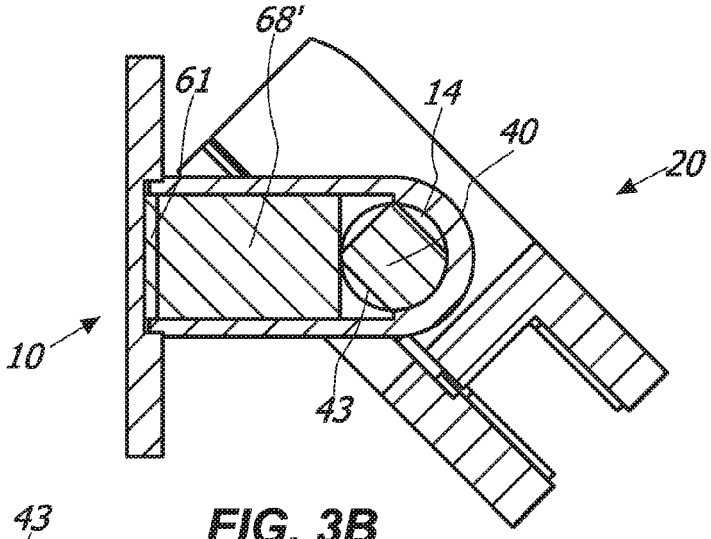

FIG. 3B

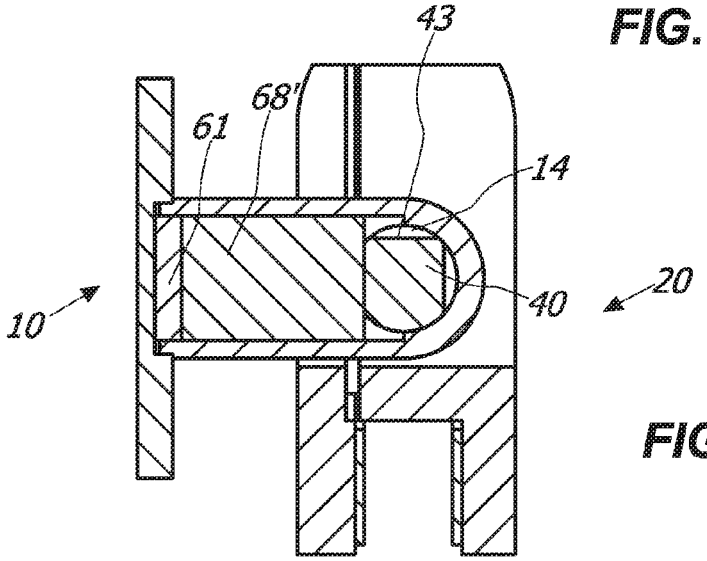

FIG. 3C

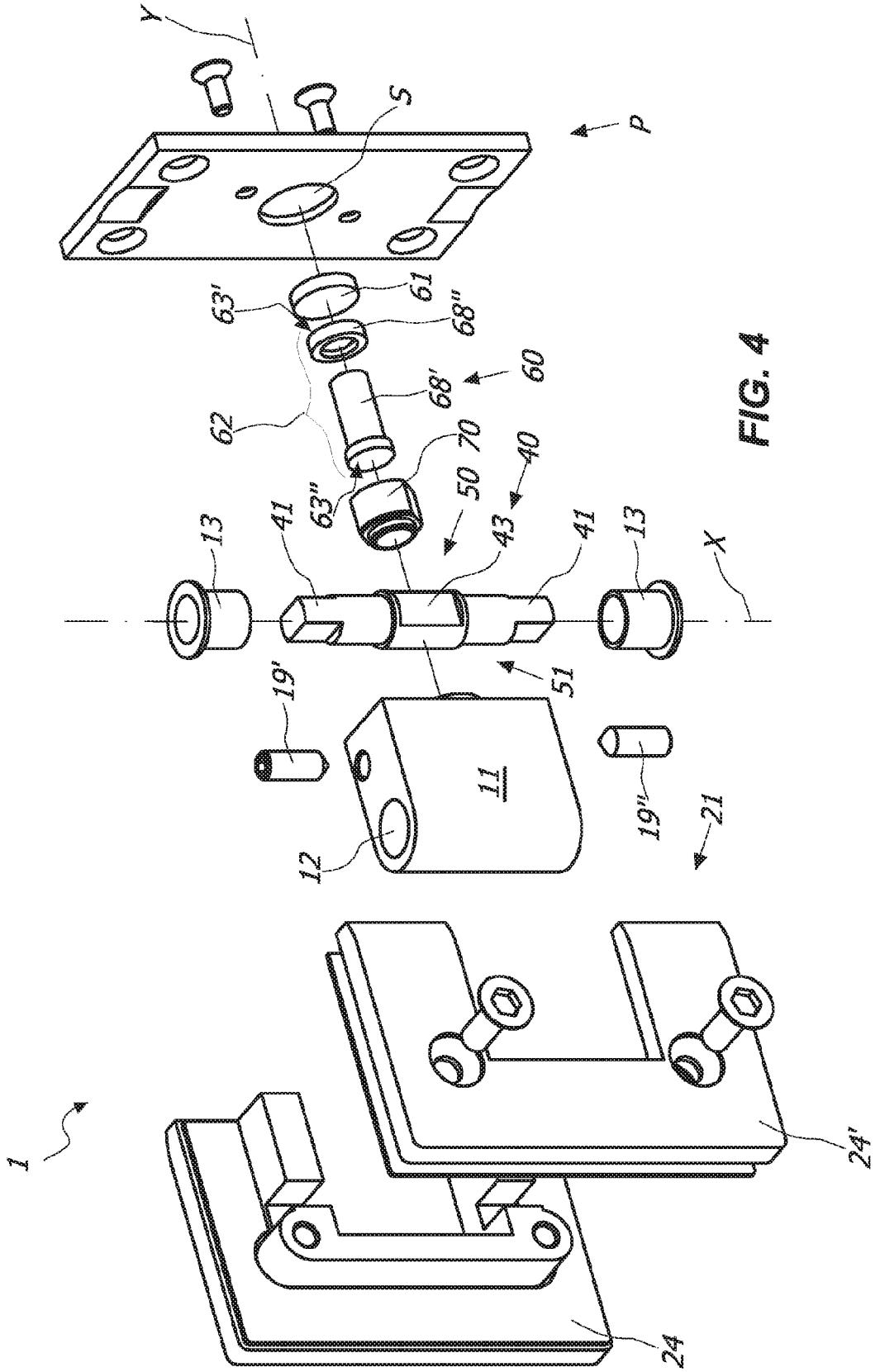

FIG. 4

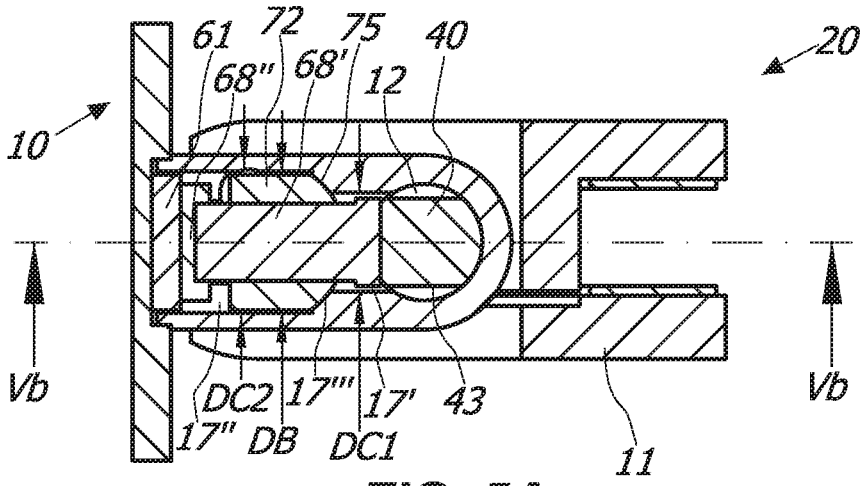
61
72
75
40
20
68''
68'
12
10
Vb
Vb
DC2
17'''
17'
43
17''
DB
DC1
11

FIG. 5A

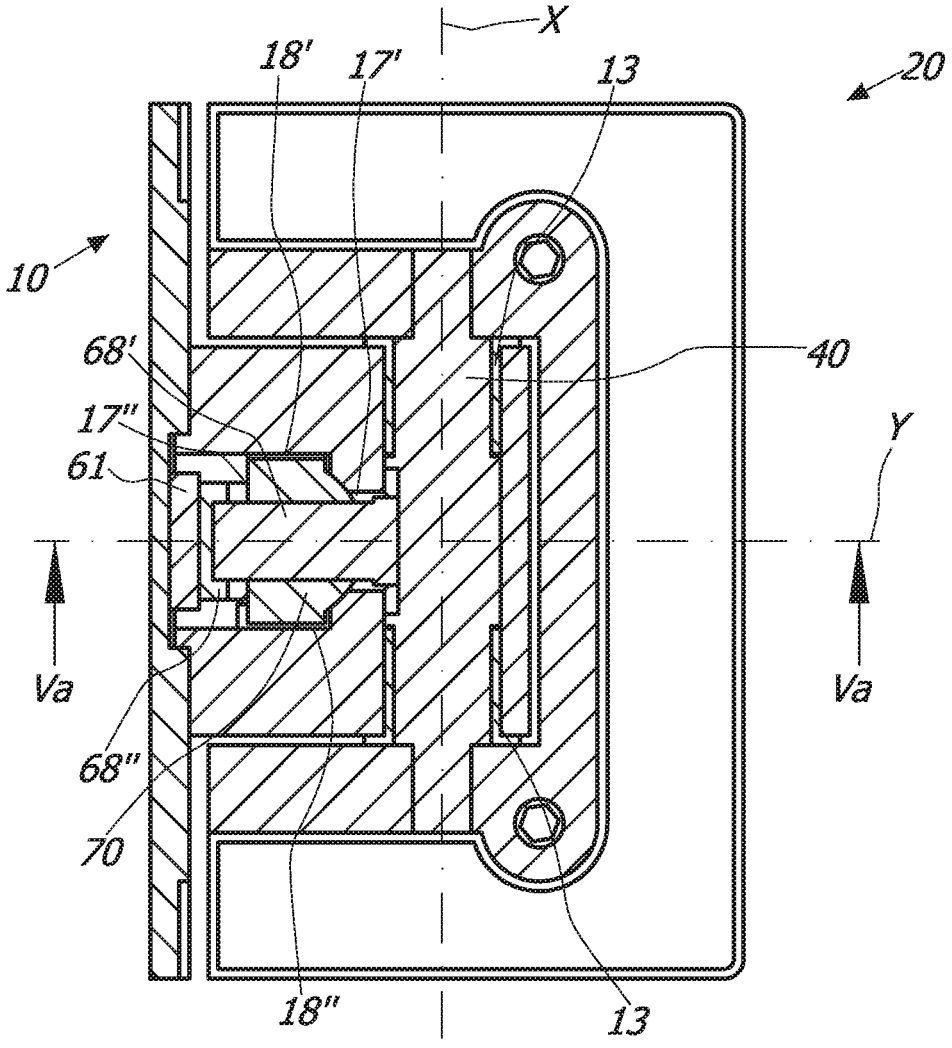
X
18'
17'
13
20
10
68'
40
17''
61
Y
Va
Va
68''
70
18''
13

FIG. 5B

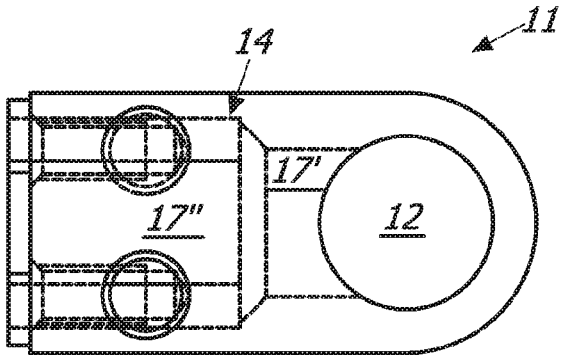

FIG. 6A

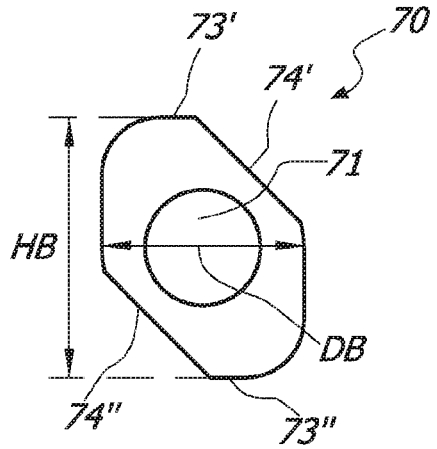

FIG. 7

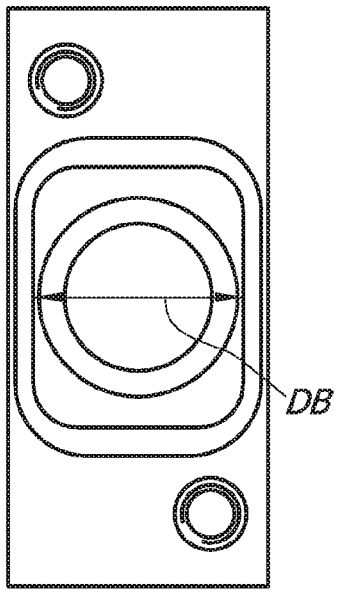

FIG. 6C

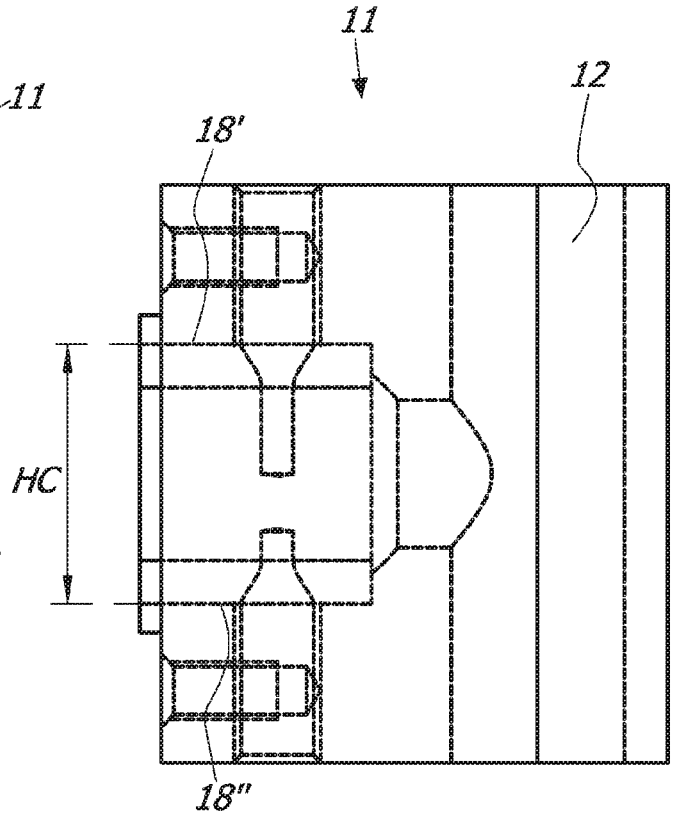

FIG. 6B

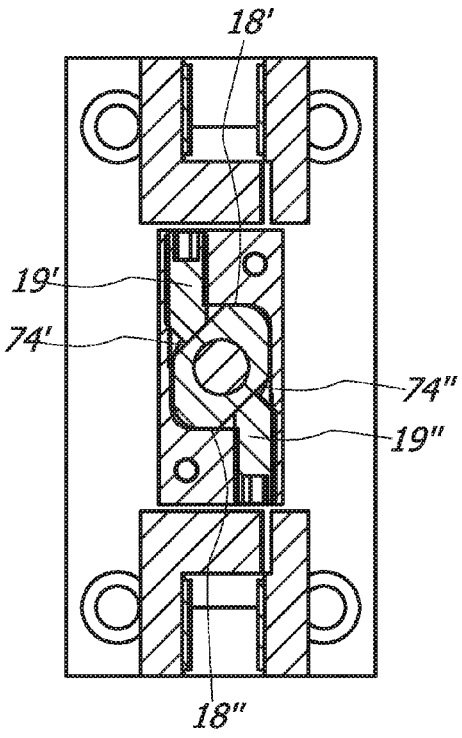

FIG. 8B

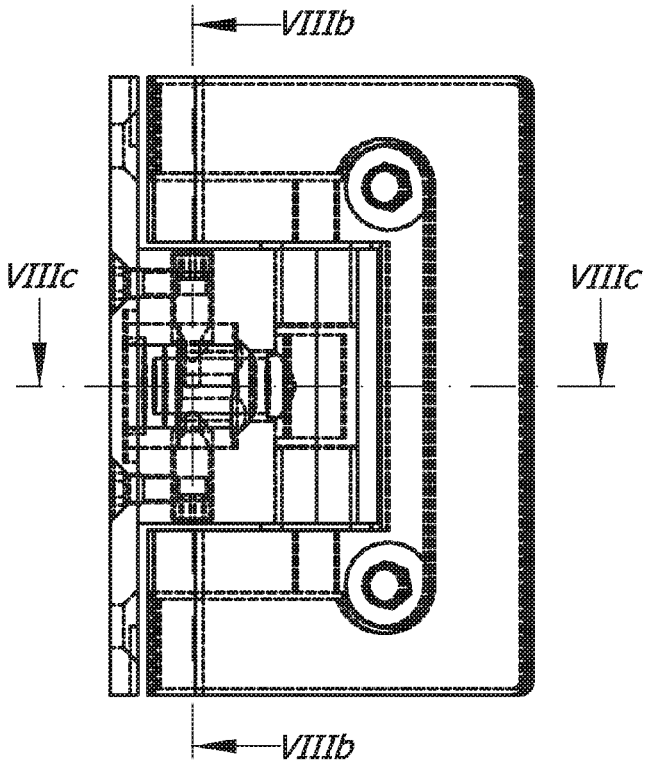

FIG. 8A

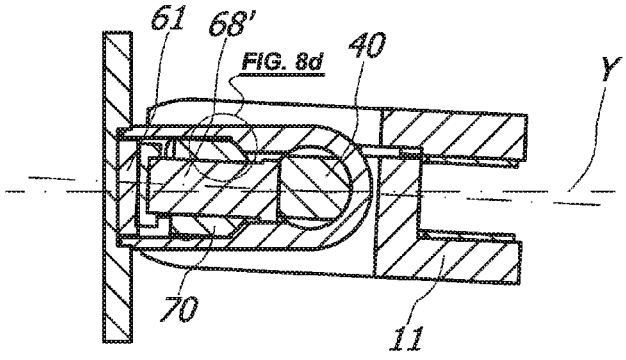

FIG. 8C

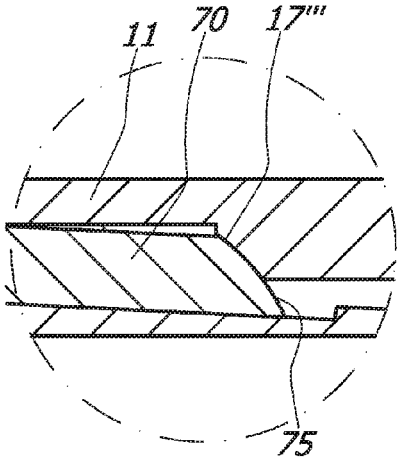

FIG. 8D

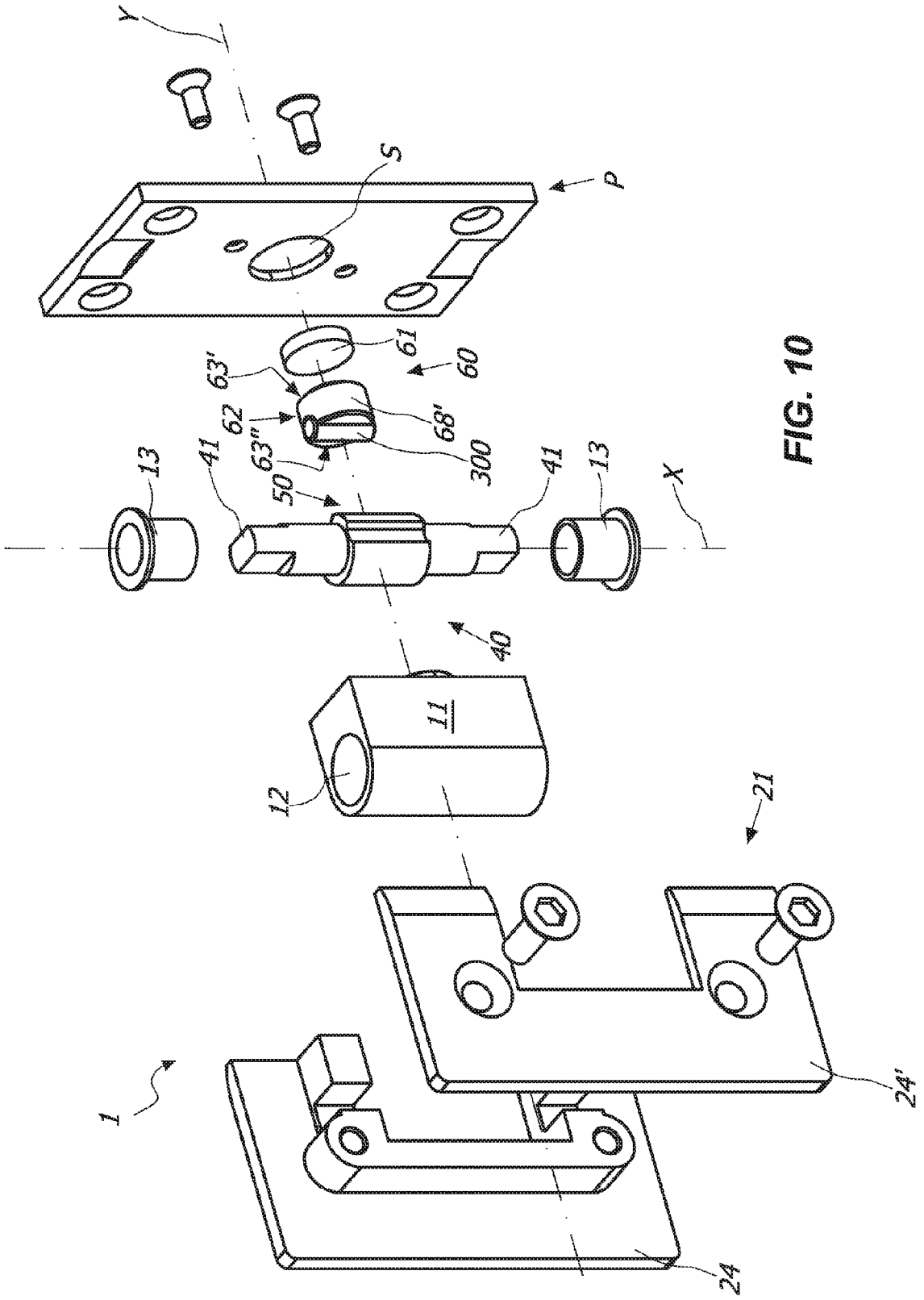

FIG. 10

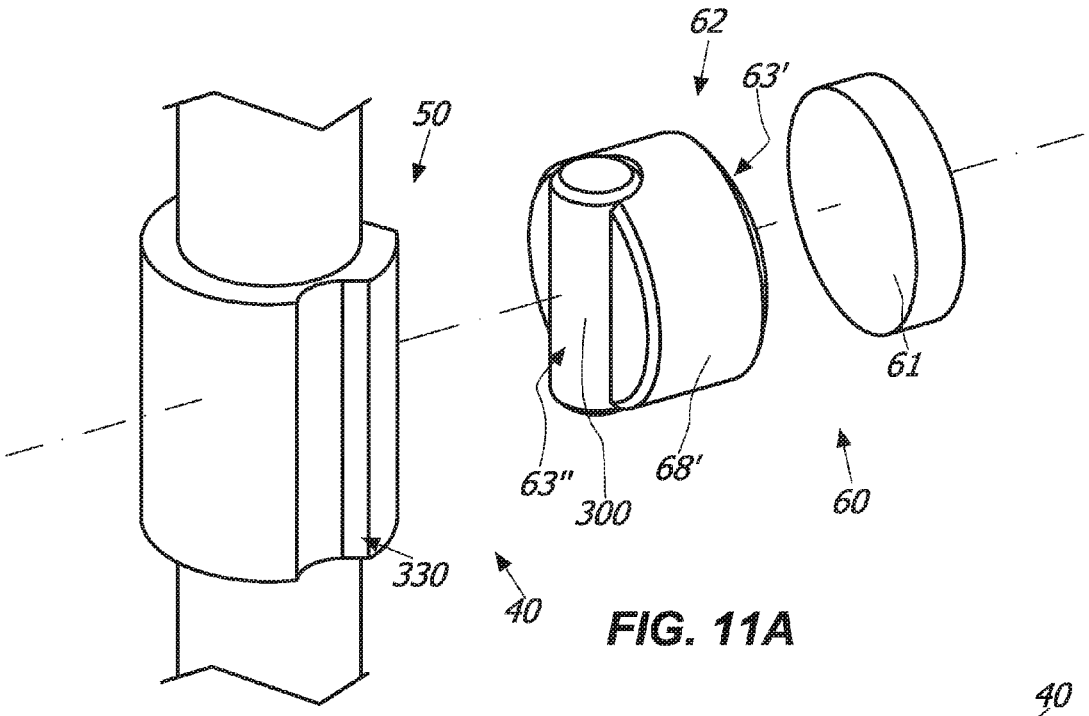

FIG. 11A

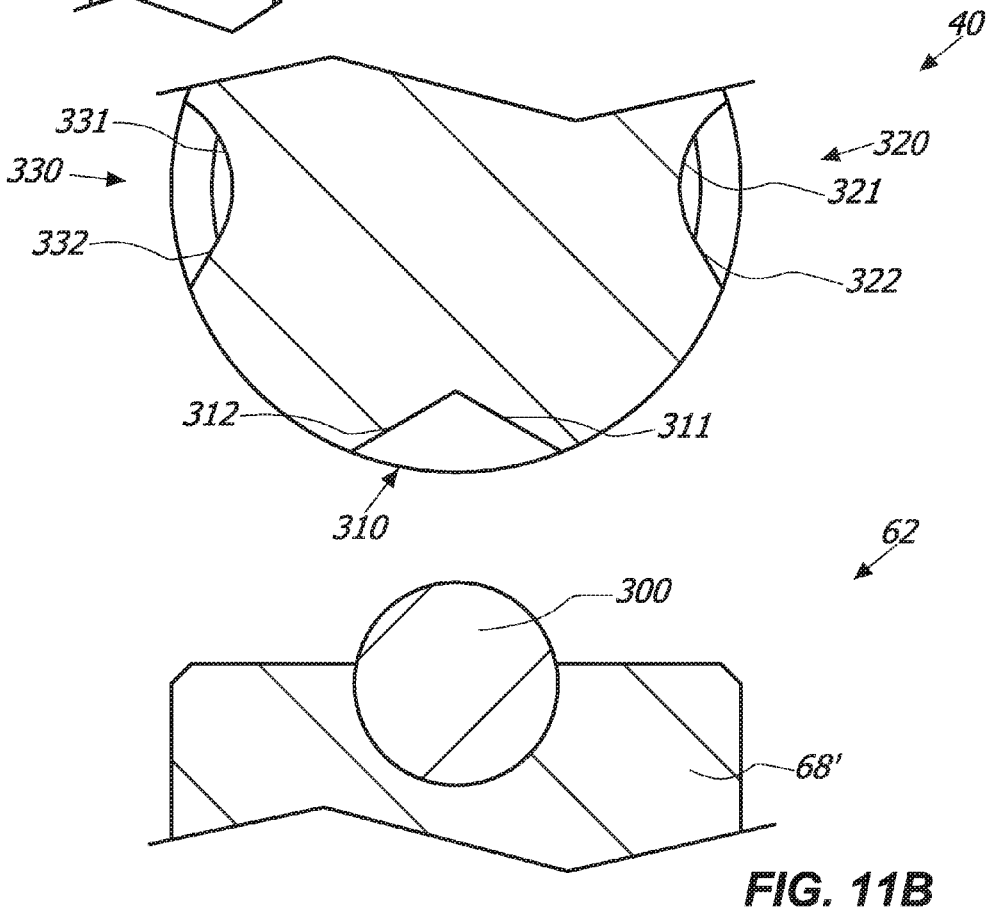

FIG. 11B

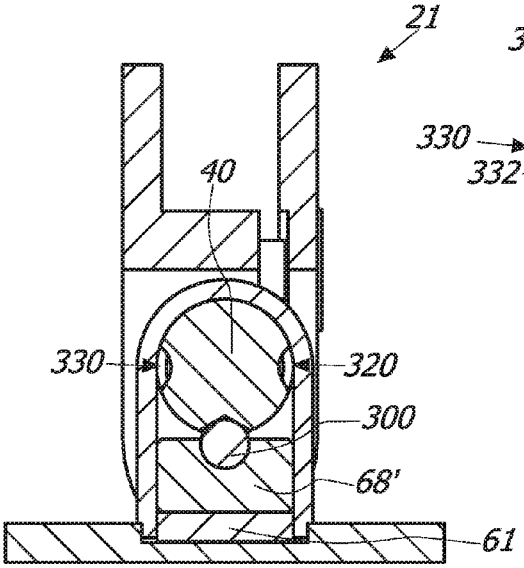

FIG. 12A

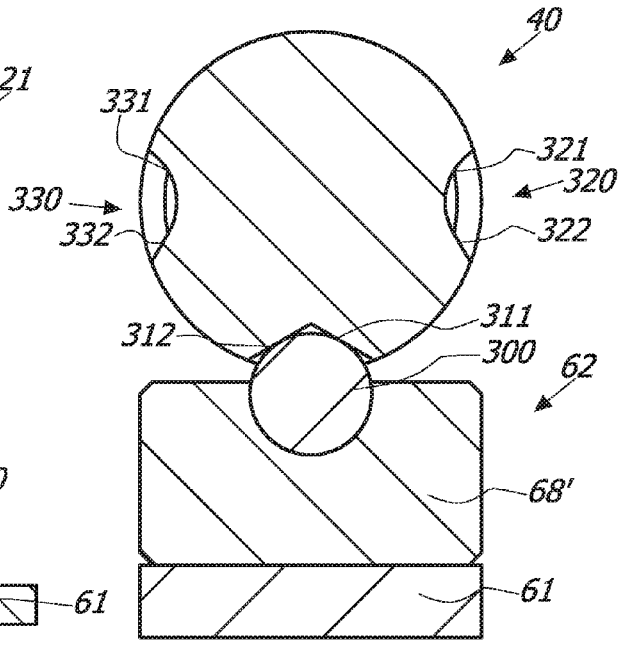

FIG. 12B

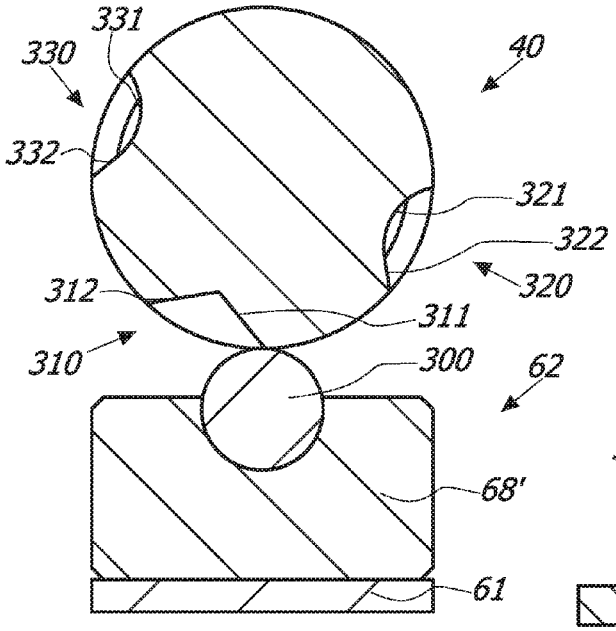

FIG. 13B

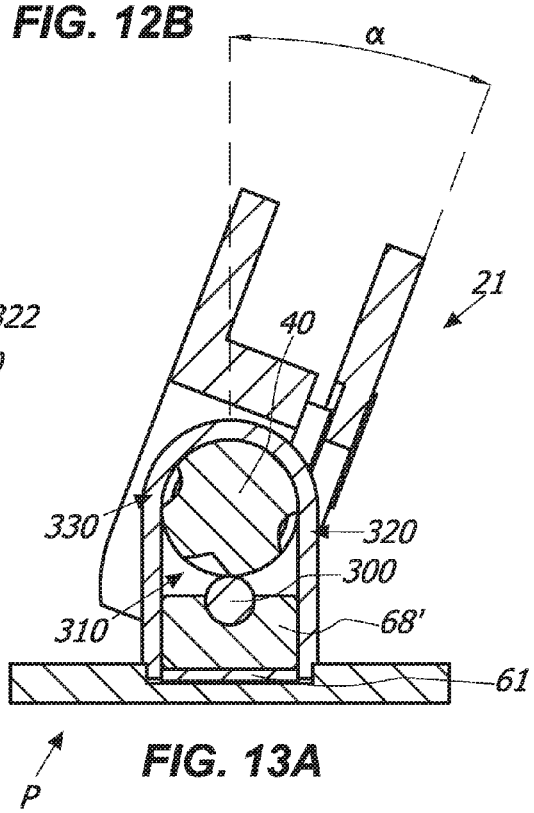

FIG. 13A

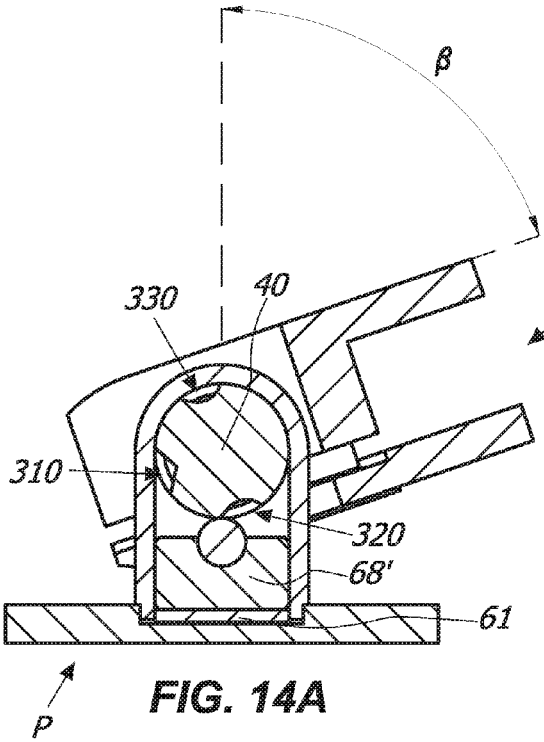

FIG. 14A

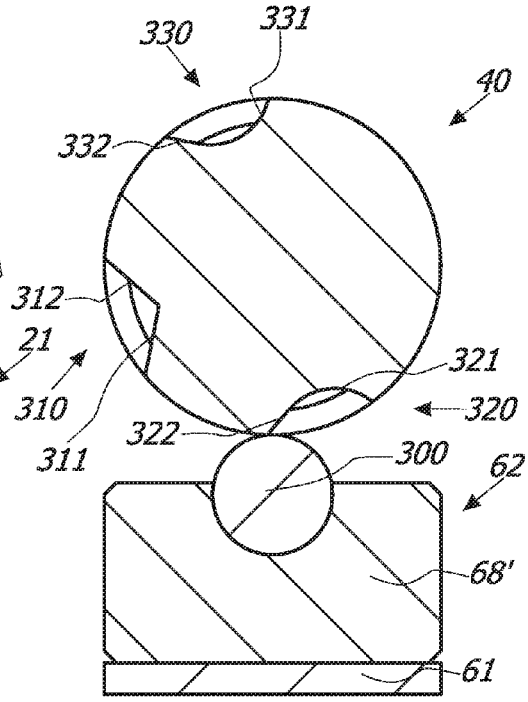

FIG. 14B

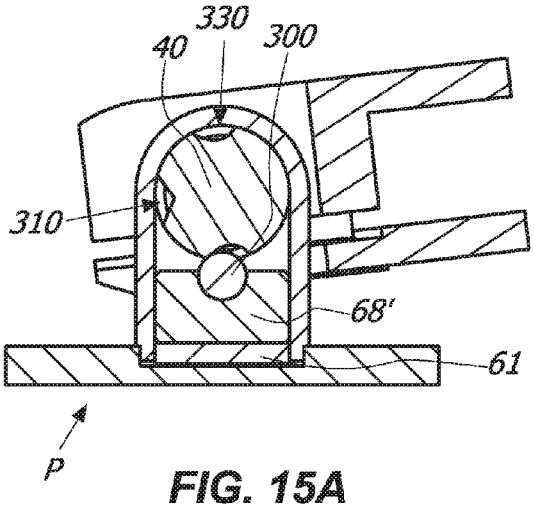

FIG. 15A

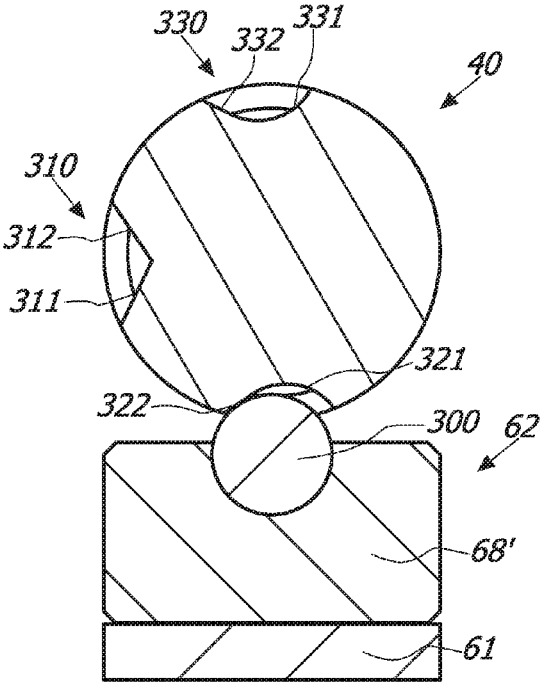

FIG. 15B

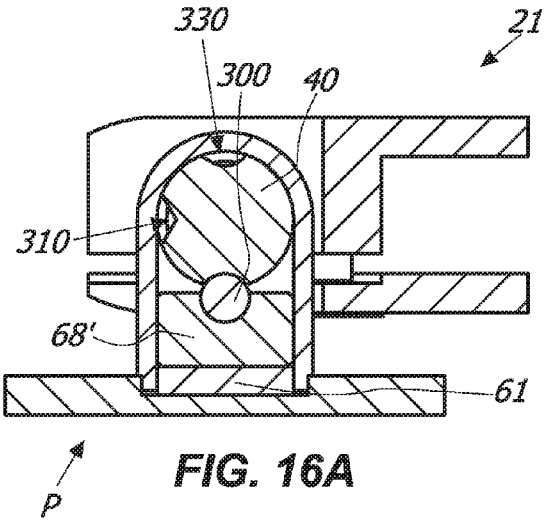

FIG. 16A

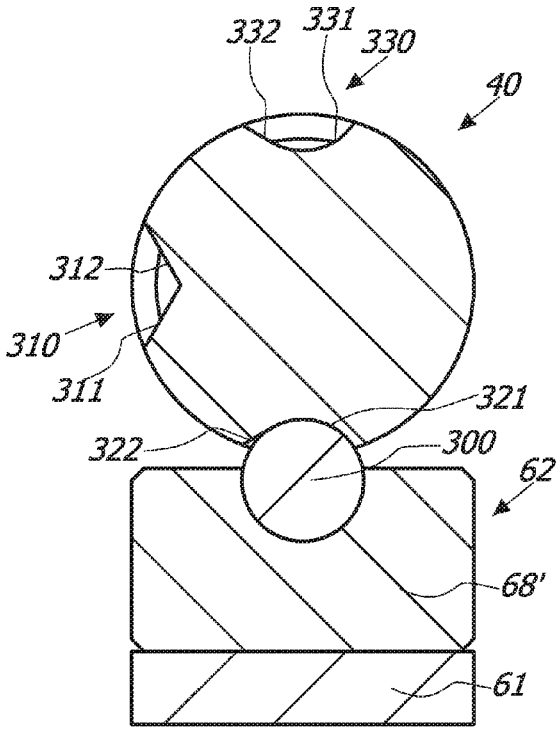

FIG. 16B

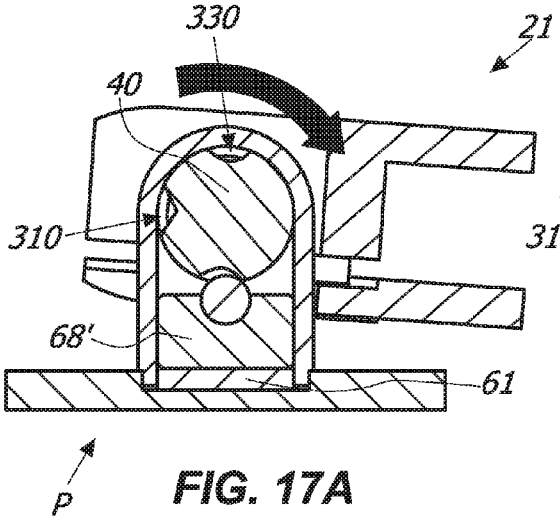

FIG. 17A

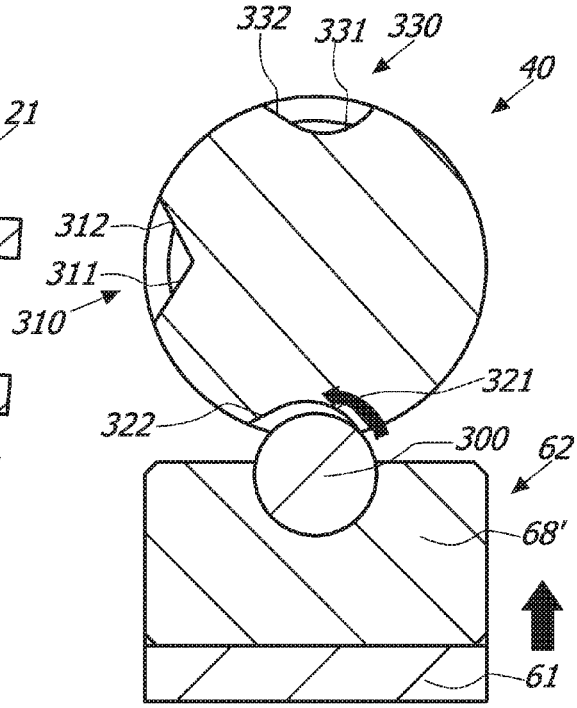

FIG. 17B

HINGE FOR THE CONTROLLED ROTATABLE MOVEMENT OF A DOOR, IN PARTICULAR A GLASS DOOR

FIELD OF INVENTION

The present invention is generally applicable to the technical field of the closing or damping/control hinges, and particularly relates to a hinge for the controlled rotatable movement of a door, in particular a glass door.

BACKGROUND OF THE INVENTION

As known, the hinges for glass doors generally comprise a movable element to be fixed to the door, which movable element is hinged on a fixed element, fixed to a support frame.

An example of such known hinges is shown in the document DE29618578U, which shows a hinge in which the door once opened is automatically closed by swinging several times around the closed position.

The absence of control makes this hinge extremely dangerous, because during the swing the door could hit an object or a person, thus breaking. It is apparent that in the case a person is close to the door, such a break may more or less seriously hurt him.

Moreover, this known hinge tends to lose the starting position and/or to misalign.

SUMMARY OF THE INVENTION

An object of the present invention is to overcome at least partly the above mentioned drawbacks, by providing a hinge having high performances, simple construction and low cost.

Another object of the invention is to provide a hinge which allows controlling the movement of the door upon its opening and/or its closing.

Another object of the invention is to provide a strong and reliable hinge.

Another object of the invention is to provide a hinge having extremely small dimensions.

Another object of the invention is to provide a hinge that has a minimum number of constituent parts.

Another object of the invention is to provide a hinge suitable to maintain the exact closing position during time.

Another object of the invention is to provide a hinge that is safe.

Another object of the invention is to provide a hinge that is easy to install.

Another object of the invention is to provide a hinge that simplifies the operations of maintenance and/or replacement thereof.

Another object of the invention is to provide a hinge which allows a simple adjustment of the door to which it is connected.

These objects, as well as other which will appear clearer hereafter, are fulfilled by a hinge having one or more of the features herein disclosed, claimed and/or shown.

Advantageous embodiments of the invention are defined in accordance with the dependent claims.

BRIEF DESCRIPTION OF THE DRAWINGS

Further features and advantages of the invention will appear more evident upon reading the detailed description of some preferred, non-exclusive embodiments of a hinge **1**, which is described as non-limiting examples with the help of the annexed drawings, in which:

FIG. **1** is an exploded view of a first embodiment of the hinge **1**;

FIGS. **2**A and **2**B are perspective views of the embodiment of the hinge **1** of FIG. **1** respectively in a closed and open position;

FIGS. **3**A, **3**B and **3**C are sectioned views of the embodiment of the hinge **1** of FIG. **1** respectively in a closed, partly open and fully open position, the section being taken along a plane IIIa-IIIa;

FIG. **4** is an exploded view of a further embodiment of the hinge **1**;

FIGS. **5**A and **5**B are sectioned views of the embodiment of the hinge **1** of FIG. **4** in the closed position, the sections being taken along planes Va-Va and Vb-Vb;

FIGS. **6**A, **6**B and **6**C are respective top, side and front views of the embodiment of the hinge **1** of FIG. **4**;

FIG. **7** is a front view of the bushing **70** of the embodiment of the hinge **1** of FIG. **4**;

FIGS. **8**A, **8**B, **8**C and **8**D are respective side, sectioned along a plane VIIIb-VIIIb, sectioned along a plane VIIIc-VIIIc and enlarged views of the embodiment of the hinge **1** of FIG. **4** in an operative configuration;

FIG. **9** is a sectional view of an alternative configuration of the pivot **40** and the pushing cylinder **68'**, equivalent to the one shown in FIGS. **3**A, **3**B and **3**C;

FIG. **10** is an exploded view of a further embodiment of the hinge **1**;

FIGS. **11**A and **11**B are respectively perspective and partly cut sectional views of some details of the embodiment of the hinge **1** of FIG. **10**;

FIG. **12**A is a sectional view of the embodiment of the hinge **1** of FIG. **10** in a first operating step;

FIG. **12**B is an enlarged view showing the relative position of the cam means **50** of FIG. **12**A, the pushing member **68'** and the elastic counteracting element **61**;

FIG. **13**A is a sectional view of the embodiment of the hinge **1** of FIG. **10** in a second operating step;

FIG. **13**B is an enlarged view showing the relative position of the cam means **50** of FIG. **13**A, the pushing member **68'** and the elastic counteracting element **61**;

FIG. **14**A is a sectional views of the embodiment of the hinge **1** of FIG. **10** in a third operating step;

FIG. **14**B is an enlarged view showing the relative position of the cam means **50**, the pushing member **68'** and the elastic counteracting element **61** of FIG. **14**A;

FIG. **15**A is a sectional view of the embodiment of the hinge **1** of FIG. **10** in a fourth operating step;

FIG. **15**B is an enlarged view showing the relative position of the cam means **50**, the pushing member **68'** and the elastic counteracting element **61** of FIG. **15**A;

FIG. **16**A is a sectional views of the embodiment of the hinge **1** of FIG. **10** in a fifth operating step;

FIG. **16**B is an enlarged view showing the relative position of the cam means **50**, the pushing member **68'** and the elastic counteracting element **61** of FIG. **16**A;

FIG. **17**A is a sectional views of the embodiment of the hinge **1** of FIG. **10** in a sixth operating step;

FIG. **17**B is an enlarged view showing the relative position of the cam means **50**, the pushing member **68'** and the elastic counteracting element **61** of FIG. **17**A.

DETAILED DESCRIPTION OF SOME PREFERRED EMBODIMENTS

With reference to the above figures, the hinge according to the invention, generally indicated **1**, is particularly useful for the rotatable possibly controlled movement during opening

and/or closing of a door, in particular a glass door, which may be anchored to a stationary support structure, such as a wall or a frame.

The embodiments of hinges **1** herein shown are adapted to be mounted to a frame of a glass door through a plate P. The embodiment shown in FIGS. **1** to **3**C differs from the one shown in FIGS. **4** to **8**D for the fact that the latter has means for adjusting the position of the door when the same is in closed position. The embodiment shown in FIGS. **10** to **17**B differs from the others for the shape of the cam means **50** and the follower means **60**.

Conveniently, the hinge **1** may include a fixed element **10**, which may be fixed to the stationary support, on which a movable element **20** is pivoted to rotate about a longitudinal axis X, which may be substantially vertical, between an open position, shown for example in FIGS. **2**B, **3**B and **3**C and a closed position, shown for example in FIGS. **2**A and **3**A.

Advantageously, the fixed element **10** may include a box-shaped hinge body **11** anchored to the stationary support, while the movable element **20** may include means **21** for fixing to the glass door. In particular, in a per se known manner, the fastening means **21** may be defined by a pair of clamps **24**, **24'** adapted to mutually cooperate to clamp a glass door.

Suitably, the hinge body **11** may include a passing-through seat **12** defining the axis X within which is inserted with minimal clearance the pivot **40**, which may be connected to the fixing means **21**.

The pivot **40** may have both ends **41** mutually connected with the fixing means **21**. In this way, the pivot **40** is unitary movable with the door between the open and closed positions.

Suitably, at the ends of the passing-through seat **12** of the box-shaped body **11** respective anti-friction elements **13** may be placed, such as bushings.

This allows the movable element **20** to rotate about the axis X with minimum friction, so that the hinge **1** is able to support even very heavy doors.

The hinge body **11** may internally include a working chamber **14** defining a second axis Y which is substantially perpendicular to the first axis X defined by the passing-through seat **12** for the pivot **40**.

Suitably, the pivot **40** may include cam means **50** rotating around the axis X, while the working chamber **14** may include follower means **60** interacting with the former to slidably move along the axis Y between a first and a second end-stroke position, shown for example in FIGS. **3**A and **3**B.

The follower means **60** may include an elastic counteracting element adapted to elastically oppose the pushing force imparted by the cam means. As non-limiting example, the elastic counteracting element may include, respectively may consist of, a spring, a nitrogen cylinder or a portion of polymeric material.

In a preferred but not exclusive embodiment of the hinge **1**, the elastic counteracting element may consist of an elastomer body **61**, which may be plate-shaped, disk-shaped or cylindrical-shaped.

Advantageously, the elastomer body **61** may be made of a polyurethane elastomer of the compact type, for example Vulkollan®. Suitably, the elastomer may have a Shore A hardness of 50 ShA to 95 ShA, preferably of 70 ShA to 90 ShA. More preferably, the elastomer body **61** may have a Shore A hardness of 80 ShA.

The use of the elastomer body **61** in place of the classic spring allows for secure stopping of the glass door without oscillations around the closed position.

Therefore, the hinge **1** is particularly safe, economical and long lasting in time. Moreover, the hinge **1** requires minimum maintenance and is extremely easy to install.

In the embodiments herein shown, the elastomer body **61** is used as urging member, in order to urge each towards the other the cam means **50** and the follower means **60** and to maintain the latter in the stop door positions, as better explained later.

Suitably, the elastomer body **61** may have discoidal shape, and may be housed in a seat S of the plate P.

In fact the plate P, in addition to allowing the connection of the hinge **1** to the stationary support structure, also acts as closing cap for the working chamber **14**.

Moreover, the follower means **60** may advantageously include an interface element **62** having a first end **63'** which interacts with the elastic counteracting element **61** and a second end **63"** interacts with the cam means **50**.

In the embodiment shown in FIGS. **1** to **3**C and **10** to **17**B, the interface element **62** may be a single piece of generally cylindrical or discoidal shape, and configured as a pushing member **68'**.

In the embodiment shown in FIGS. from **4** to **8**D, the interface element **62** may be composed of two pieces, a pushing cylinder **68'** and a pressure disc **68"** inserted in a bushing **70**, the function of which is better explained later.

Suitably, the pivot **40** may include the cam means **50**, so that the latter rotate unitary with the former around the axis X. The cam means **50** may in turn include one or more cam elements adapted to interact with the follower means **60**.

In a preferred but not exclusive embodiment, the cam means may be defined by a plurality of flat faces **43** formed at the central portion of the pivot **40**.

The relative angle between the flat faces of the cam means determines the stop positions of the door.

In particular, in the embodiments shown herein, the flat faces **43** may be three, mutually perpendicular each other to define an equal number of stop door positions, in the closed position and the open ones in both possible directions.

In fact, the elastomer body **61** pushes the pushing cylinder **68'** against the flat faces **43** formed at the central portion of the pivot **40**, so as to maintain the relative door open or closed positions.

To this end, the interface element **62** may have the second end **63"** that includes a substantially planar operating surface **66** susceptible to come in contact with the substantially planar faces **43**.

In this way, in the stop positions of the door the substantially planar operating surface **66** is parallel to the flat face **43** by which it interacts, in order to ensure the stability of the position.

It is understood that in this document the terms “flat face” and “planar surface” and their derivatives indicate faces or surfaces whose geometry, even if not actually flat or planar, is equivalent thereto.

FIG. **9** shows a flat face **43** and a planar operating surface **66** which, although not actually flat, are equivalent to faces or surfaces flat or planar. Indeed, their geometry is such that the edges **45'**, **45"** of the face **43** defines a flat surface resting on the operating surface **66**.

It is understood that any other geometry or configuration adapted to provide a flat surface or face falls within the scope of protection of the appended claims.

Advantageously, in the stop positions the flat faces **43** may be mutually in contact with the substantially planar operating surface **66** throughout all its width, as shown for example in FIGS. **3**A and **3**C.

To ensure the stability of the stop position even in the event of accidental knocks to the door, the length Les of the contact surface between the substantially planar operative surface **66** and the substantially flat faces **43** may be slightly less than the diameter Dp of the pivot **40**.

Suitably, the ratio Les/Dp between the length Les above and the diameter Dp of the pivot **40** may be not less than 0.8, and preferably equal to or greater than 0.85.

Due to this feature, the hinge **1** is extremely safe, in particular in those applications in which there is a danger that an unwary user inadvertently bumps the door. In fact, in the case of glass door this may result in the breaking of the door and the consequent injury of the user.

To maximize this effect, in a preferred but not exclusive embodiment, between the substantially planar faces **43** a connecting portion **44** may be interposed having a width Lpc substantially less than the one Lsc of the same flat faces **43**. Preferably, the connecting portion **44** may have non-planar shape, for example a rounded shape.

This results in the maximum possible compression of the elastic counteracting element **61**. In other words, to move from a stop position to another the user must exert a relatively high force on the door, thus minimizing the risk that small bumps may move the door with the above consequences.

Suitably, the ratio between the width Lpc of the connection portion **44** and the one Lsc of the flat faces **43** may be not more than 0.2, and preferably less than 0.15.

In another preferred but not exclusive embodiment, the interface element **62** may be configured as a pushing member **68'** and include a protrusion **300**, having a generally hemispherical shape. On the other hand, the cam means **50** may include a plurality of seats **310**, **320**, **330** each corresponding to a stop position of the door.

More in particular, the seats **310**, **320**, **330** are able to receive the positions **300** to stop the door in the stop positions.

Suitably, the seat **310** may correspond to the closed door position, while the seats **320**, **330** may correspond to the open door positions. Advantageously, the latter may be mutually opposite with respect to the closed door position.

In a preferred but not exclusive embodiment, the seat **310** corresponding to the closed door position may have a generally "V"-shape with two consecutive planes **311**, **312** angled each other with predetermined angle.

In this way, as particularly shown in FIG. **13**A, the sliding of the hemispherical protrusion **300** on the planes **311**, **312** upon the rotation of the door is simplified, so as to ensure the automatic closing of the door starting from a predetermined angle α, for example 20°.

At the same time, user can rotate the door from the closed door position in both opening directions.

To maximize this effect, the angle between the planes **311**, **312** may be at least 90°, preferably at least 110°. In a preferred but not exclusive embodiment, the angle between the planes **311**, **312** may be 120°.

Moreover, each of the seats **320**, **330** corresponding to the open door positions may advantageously have two consecutive portions **321**, **322**; **331**, **332** having different shape.

The first portions **322**; **332** may be generally flat, while the second portions **321**; **331** may be countershaped with respect to the shape of the protrusion **300**, and in particular may be hemispherical.

In this way, the first flat portions **322**; **332** may promote the sliding of the projection **310** thereon to convey it towards the second portions **321**; **331**, suitable to stop the door.

In this way, as particularly shown in FIG. **14**A, the automatic opening of the door starting from a predetermined angle for example 70°, is ensured.

As particularly shown in FIGS. **15**A and **15**B, the first flat portions **322**; **332** act as pilot members for the second hemispherical portions **321**; **331**, an that the insertion of the protrusion **300** in the latter takes place without noise.

Advantageously, the first flat portions **322**; **332** may be substantially perpendicular to the planes **312**, **311**.

Moreover, thanks to the above configuration the door may be rotated from the stop position only in one direction. In other words, the rotation in the other direction is prevented.

Indeed, as shown in FIG. **17**B, if a user attempts to further rotate the door, the momentum caused by the elastic counteracting element **61** opposes this force, which momentum urges the one against the other the protrusion **300** and the second portions **321**; **331**.

Suitably, the elastic counteracting element **61** may be configured so as to allow a further slight rotation of the door after the stop position in the door open position. To this end, the elastic counteracting element **61** after this minimum rotation can reach the position of maximum compression.

This absorbs the shock undergone by the door upon the reaching of the stop position. This configuration is particularly advantageous in the case of glass door, which in the case of abrupt shock could be damaged or broken.

The embodiment shown in FIGS. **10** to **17**B and described above is particularly advantageous with the above described elastic counteracting element **61** made of elastomer.

In fact, in the latter a minimum stroke corresponds to a very high strength.

Therefore, suitably precompressing the elastic counteracting element **61** in the working chamber **14** the strength of the hinge **1** is maximized.

Also, the elastic counteracting element **61** made of elastomer maximizes the effect of stopping the rotation, as described above.

The shape of the cam means **50** determines the stroke of the elastomer body **61**. In particular, the cam element may be configured so that the stroke can be of 1 mm to 5 mm, and preferably of 1 mm to 3 mm.

In the embodiment of the hinge **1** shown in FIGS. **4** to **8**D is possible to adjust the position of the movable element **20** in the closed door position.

For this purpose, a bushing **70** may be provided with a central hole **71** which houses the pushing cylinder **68'**. The bushing **70** may include a tubular portion **72** having an outer diameter DB and a height HB. The bushing **70** may further have substantially flat upper and lower surfaces **73'**, **73"**, and slanted peripheral portions **74'**, **74"**.

On the other hand, the working chamber **14** may include a first tubular portion **17'** having a first inner diameter DC**1** and a second portion **17"** of generally rectangular shape and transverse dimension DC**2** and height HC.

The bushing **70** may be inserted into the working chamber **14** with the tubular portion **72** placed in correspondence of the second portion **17"** of the same working chamber **14**.

The outside diameter DB of the portion **72** of the bushing **70** may be slightly less than the inside diameter DC**2** of the portion **17"** of the working chamber **14**. The height HB of the portion **72** of the bushing **70** may be substantially equal to the height HC of the second portion **17"** of the working chamber **14**.

The connecting portion **17'"** between the two portions **17'** and **17"** of the working chamber **14** may be suitably rounded, as well as the corresponding operating portion **75** of the bushing **70**.

Thanks to this configuration, the bushing **70** is free to transversely move once inserted in the working chamber **14**. The stroke of this movement is defined by the difference

between the outer diameter DB of the portion **72** of the bushing **70** and the inner diameter DC**2** of the portion **17"** of the working chamber **14**. During this movement, the bushing **70** is horizontally guided by the sliding of the substantially flat upper and lower surfaces **73'**, **73"** on the walls **18'**, **18"** of the portion **17"** of the working chamber **14**, which is also flat.

To adjust the movement, adjusting screws **19'**, **19"** may be provided acting on the slanted portions **74'**, **74"**. In practice, the adjusting screws **19'**, **19"** act in a substantially vertical direction, and the inclined planes defined by the slanted portions **74'**, **74"** transmit the horizontal component of the pushing force to the bushing **70**, thus causing the shift thereof in the portion **17"** the working chamber **14**.

Furtherly, the connecting portions **17'"** of the working chamber **14** and the corresponding operating portion **75** of the bushing **70** cooperate with each other to allow the partial rotation of the bushing **70**, in such a way as to vary the inclination of the axis Y, and therefore the closed door position, as particularly shown in FIG. **8**C.

From the above description, it is apparent that the hinge **1** fulfils the intended objects.

The hinge **1** is susceptible to many changes and variants. All particulars may be replaced by other technically equivalent elements, and the materials may be different according to the needs, without exceeding the scope of the invention defined by the appended claims.

The invention claimed is:

1. A hinge for coupling a door and a stationary support structure, the hinge comprising:

a fixed element to be coupled to the stationary support structure; and

a movable element to be coupled to the door, the fixed element and the movable element being rotatably coupled each other to rotate about a first longitudinal axis between one or more open positions and a closed position;

wherein one of said movable element and fixed element includes a hinge body, the other of said movable element and fixed element including a pivot defining said first axis, the pivot including a cam member rotating relative to the hinge body about the first axis, said hinge body including at least one working chamber defining a second longitudinal axis perpendicular to said first axis, said at least one working chamber including a follower member interacting with said cam member, the follower member sliding along said second axis between a first and a second end stroke position;

wherein said follower member includes at least one elastic counteracting element and at least one interface element having a first end interacting with said at least one elastic counteracting element and a second end interacting with said cam member, said at least one elastic counteracting element including an elastomer body;

wherein said cam member includes a plurality of flat faces parallel to said first axis, said flat faces being perpendicular to each other, said second end of said at least one interface element including at least one planar operating surface in contact engage with each one of said flat faces of said cam member along a respective contact surface;

wherein the contact surfaces between said at least one planar operating surface and said flat faces have a respective first width, said pivot having a diameter, the ratio between the first widths of said contact surfaces and the diameter of said pivot being not less than 0.8.

2. The hinge according to claim **1**, further including a bushing transversely movable within said working chamber with a tubular portion faced to said cam member and an operating portion faced to said elastic counteracting element for cooperating with a corresponding guide surface of said working chamber, said bushing further including a central hole for housing at least partly said interface element and a pair of peripheral slanted portions.

3. The hinge according to claim **2**, further including a pair of adjusting screws acting on said slanted portions to move transversely said bushing in said working chamber so as to vary the angle of said central hole with respect to said second axis.

4. The hinge according to claim **3**, wherein said operating portion of said bushing and said guide surface of said working chamber are both rounded.

5. The hinge according to claim **4**, wherein said working chamber includes a first cylindrical portion facing said cam member having a predetermined inner diameter and a generally rectangular-shaped second portion facing said at least one elastic counteracting element having a predetermined transverse dimension and height, said bushing being placed within said second portion of said working chamber.

6. The hinge according to claim **5**, wherein said tubular portion of said bushing has a predetermined outer diameter and height, said second portion of said working chamber including a pair of substantially flat upper and lower walls faced to each other, said tubular portion of said bushing having upper and lower substantially flat surfaces susceptible to transversely slide along said substantially flat upper and lower walls of said second portion of said working chamber in response to the action of a user on said adjusting screws.

7. The hinge according to claim **6**, wherein the height of said tubular portion of said bushing is substantially equal to the height of said second portion of said working chamber, the outer diameter of said tubular portion of said bushing being slightly lower than said transverse dimension of said second portion of said working chamber for allowing the transverse movement of said bushing.

8. The hinge according to claim **7**, wherein said working chamber includes a connecting portion interposed between said first cylindrical portion and second cylindrical portion which includes said guide surface.

9. The hinge according to claim **1**, wherein said elastomer is a compact polyurethane.

10. The hinge according to the claim **1**, wherein said elastomer has a Shore A hardness of 50 ShA to 95 ShA.

11. The hinge according to claim **1**, wherein said ratio between the lengths of said contact surfaces and the diameter of said pivot is equal to or greater than 0.85.

12. The hinge according to claim **1**, wherein said fixed element includes said hinge body, said movable element including said pivot, said cam member being made in the central portion of the pivot.

13. A hinge for coupling a door and a stationary support structure, the hinge comprising:

a fixed element to be coupled to the stationary support structure; and

a movable element to be coupled to the door, the fixed element and the movable element being rotatably coupled each other to rotate about a first longitudinal axis between one or more open positions and a closed position;

wherein one of said movable element and fixed element includes a hinge body, the other of said movable element and fixed element including a pivot defining said first axis, the pivot including a cam member rotating relative to the hinge body about the first axis, said hinge body including at least one working chamber defining a second longitudinal axis perpendicular to said first axis,

said at least one working chamber including a follower member interacting with said cam member, the follower member sliding along said second axis between a first and a second end stroke position;

wherein said follower member includes at least one elastic counteracting element and at least one interface element having a first end interacting with said at least one elastic counteracting element and a second end interacting with said cam member, said at least one elastic counteracting element including an elastomer body;

wherein said cam member includes a plurality of flat faces parallel to said first axis, said flat faces being perpendicular to each other, said second end of said at least one interface element including at least one planar operating surface in contact engage with each one of said flat faces of said cam member along a respective contact surface;

wherein the contact surfaces between said at least one planar operating surface and said flat faces have a respective first width, said pivot having a diameter, the ratio between the first widths of said contact surfaces and the diameter of said pivot being not less than 0.8;

wherein a non-flat connecting portion is interposed between each couple of consecutive flat faces, each connecting portion having a second width, the ratio between said second width and said first width being not greater than 0.2.

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